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<http://dx.doi.org/10.21278/brod70401> **ISSN 0007-215X**

eISSN 1845-5859

PSO-BASED PID CONTROLLER DESIGN FOR SHIP COURSE-KEEPING AUTOPILOT

UDC 629.5.017.3:629.3.027.2 Original scientific paper

Summary

This paper deals with autopilot proportional-integral-derivative (PID) controller design. In the literature, various available methods for PID controller have been presented. Based on the fact that the existing methods do not guarantee the optimal response of the system on the input step change, in this paper we used a valuable technique called Particle Swarm Optimization (PSO) in order to optimally design PID controller taking into account the system limitation such as the value of the rudder angle saturation. Furthermore, we have compared system response on input step and ramp change of input signal for a few PID controller parameters values obtained by using different methods known in literature. It has been proven that by applying the PSO method, it is possible to determine the optimal PID controller parameters which guarantee fast and proper response from the aspect of the minimal overshoot and the minimal settling time. The obtained results confirm the applicability and efficiency of using PSO method for optimal autopilot PID controller design.

Key words: Autopilot; PID controller; Particle Swarm Optimisation (PSO); Rudder angle

1. Introduction

During its movement, vessel is steering a certain course. Operator monitors movement and position of the ship with his senses (sight, feeling for tilt and acceleration, etc.). The status of ship's movement and position is obtained depending on the situation based on the visual perception of certain fixed points as landmarks or based on readings on instruments. Based on obtained information, a person forms a control signal and gives commands from the bridge control panel which are transmitted to the steering gear mechanism. By using steering gear mechanism, helmsman maintains existing course or changes the course (maneuvering) [1]. This type of control is based on the difference between the existing and the desired course, and the rudder mechanism is a part of the automatic control closed-loop [2].

A ship, being an autonomous and highly complex dynamic system, is composed of a series of different processes, machines and devices that lend themselves to automation. The automation of a ship's processes contributes to its higher efficiency, cutting maintenance and crew costs, prolonging its lifespan and bringing a host of other advantages.

The automatic steering system's basic task is to maintain the ship's course, i.e. to maintain a current navigational trajectory. The autopilot is a higher level of control in which the ship's course is controlled without the participation of a helmsman [2]. By constantly monitoring the actual against the desired course, an error signal is determined, and the microcomputer formulates a steering algorithm that puts the ship back on its desired course.

Proportional-integral-derivative (PID) controllers or proportional-derivative (PD) controllers are usually used in ships autopilot design [3,4]. The PID autopilot was developed with the intention of enabling a vessel to follow course as accurately as possible by decreasing the error caused by excessive deviations of the helm and by simultaneously limiting the rudder deviation in order to minimize rudder skid [5]. As an addition to the fact that a straight course is not the most economical option, it has been decided that helm control must always be optimized relative to the prevailing state of the environment and that a small bandwidth should be used in order to minimize losses. There are different designs of ship course-keeping autopilots. For example, the steering parameters for normal adaptable PID autopilots have been developed during the last three decades as specified in [6-8], and the most important among them is the performance index regarding added resistance due to imperfect steering control. Also, designs based on neural network [9,10], fuzzy logic [11-12], backstepping [13], selftuning control [14], pole placement technique (PPT) [15], extended state observer technique (ESO) [16] and similar are widely used.

As already mentioned, in [15] an analytical method for determining the PID controller parameters was presented. This method is based on the use of a symmetrical optimum and provides very simple formulation for determining PID controller parameter values. Moreover, the authors were analyzing the effect of step and ramp change of input signal and they obtained a good system response in terms of eliminating the errors in steady/stationary state. On the other hand, in [17], the application of the optimization method to determine the optimal values of PID parameters was obtained. It was considered that optimum values of the PID parameters are inside the range $\pm 10\%$ of the parameter values compared to analytical method given in [15]. Considering the mentioned references [15] and [17], it is important to point out that the steering machine limitation was not taken into account in either study.

On the other hand, a detailed description of the ship's mathematical model and steering machine limitation is presented in [18] where authors analyzed the application of extended state observer for yaw control, which as output does not give certain values of the PID parameters of the autopilot. In addition to the mentioned works, the use of Lyapunov's theory in PID controller design is reported in [19]. Specifically, using Lyapunov's theory in determining the PID controller parameters of autopilot, a very slow system response is obtained, what can be inferred from the results shown in [18]. From the comprehensive literature review in the area, it can be concluded that the above-mentioned works either use complex mathematical apparatus or provide analytical solutions that do not lead to the optimal results. Moreover, according to the researches done in [15,17] no real model of the ship has been taken into account including the steering machine limitation.

However, this paper presents an upgraded investigation on PID controller design that was previously analyzed in [15,17]. The comparison of the system response to the step change in input signal (yaw angle step change), by different methods is presented in [18]. In the available works, the authors did not compare the values of overshoot, rise time, settling time and delay time. Otherwise, these transition process parameters define the efficiency, stability and

response speed of the regulator. Based on the previous analyses, the aim of this paper is to improve the PID controller design from the perspective of better response quality with respect to limitations dictated by the components of the regulation loop (for example, hydraulic pumps, rudder angle etc.). This improvement was achieved by using the optimization technique called Particle Swarm Optimization (PSO), which is a very powerful technique, whose application can be found in a number of areas such as power converters [20], solar cells [21], electrical machines [22], power network [23], etc.

This paper is organized as follows. In Section 2 a mathematical model of a ship and PID controller is proposed. PSO technique with corresponding algorithm and its explanation is described in Section 3. Simulation results with specific values of parameters including robustness analysis are presented in Section 4. In addition, comparative analysis that includes methods from the literature and novel ones provided in this paper are given in Section 5.

2. Mathematical model

It is well known that ships are equipped with autopilots consisting of PID controllers among others, which make part of the automatic control system and their purpose is to maintain a given course of the ship. During the design of the ship, it is very important to install a control circuit and an automatic control system, i.e. autopilot, in order to relieve the helmsman.

PID-based steering course autopilot is one of the most commonly used types of autopilot for navigating the course of the ship. The course autopilot usually contains a basic algorithm for course maintenance, with or without adaptation to navigation conditions, and a maneuver controller. Setting up the parameters of the PID controller is extremely important since the parameters of the ship represent an object of the steering, i.e. the parameters of the ship dynamics change with the speed, position of the rudder, load, etc. The same is also significant due to disturbances such as winds, waves, currents, etc.

Block diagram of overall structure of ship autopilot is shown in Figure 1. In this figure, the rudder angle (or rudder deflection) is denoted with δ and ψ represents ship heading angle which is closed to its desired value ψ_d . Based on the value of heading angle error ε which appears due to external disturbances, the autopilot generates the input signal for rudder actuator [12].

Fig. 1 Overall structure of ship autopilot

The rudder actuator represents the actuation mechanism that moves the rudder to the controller commanded angle. However, the response of the rudder actuator is defined by the speed of the rudder, whereas the rudder movement is mechanically limited. The rudder is moved by hydraulic pumps, the speed of which is governed by the pump capacity and by opening the valve. Hydraulic fluid flow regulated by the swash plate of the steering machine is controlled by the telemotor system which receives signals from the autopilot controller. Detailed

description of rudder actuator can be found in [18]. According to the SOLAS Convention [24], the power of the rudder actuator must be sufficient to shift the rudder from 35° on one side, to 30° on the other side in 28 seconds.

As proposed in [15], a PID controller with an additional degree of freedom should be used to control the autopilot. Its mathematical equation is:

$$
H_c = \frac{k_c}{sT_c} (sT_c + 1) \frac{sT_c' + 1}{sT_1 + 1}
$$
 (1)

where: k_c is PID gain coefficient, T_c is PID main time constant, and T_c' and T_1 are time constant where $T_1 < T'_c < T_c$. By using this equation, the PID controller is designed by combining the pole placement method with the symmetric optimal criterion [15]. If the natural frequency ω_0 and the attenuation coefficient ξ are known, the unknown parameters of the PID can be obtained by using the following equations:

$$
k_c = \frac{\omega_0}{k_p},\tag{2}
$$

$$
T_c = \frac{2\xi + 1}{\omega_0},\tag{3}
$$

$$
T_c' = T_p \,, \tag{4}
$$

$$
T_1 = \frac{1}{(2\xi + 1)\omega_0} \,. \tag{5}
$$

Finally, the most widely used model of the ship is based on the *Nomoto* linear model [18] whose differential equation is as follows

$$
\ddot{\psi}(t) + \frac{1}{T}\dot{\psi}(t) = \frac{k}{T}\delta(t),\tag{6}
$$

where ψ and δ are earlier defined, *k* represents the static yaw rate gain and *T* is the effective yaw rate time constant. The corresponding transfer function is given by

$$
H_p = \frac{k_p}{s(sT_p + 1)}.\tag{7}
$$

Therefore, the plant has a low-order model, which contains a pure integrator, and which is characterized by a dominant time constant T_p and a gain coefficient k_p .

3. PSO algorithm

An algorithm that is based on the PSO metaheuristic belongs to the category of algorithms inspired by the swarm intelligence. Similar to bird flocking, this method is originally based on a group of particles that are flying among the search space in order to find the best position. Generally speaking, PSO algorithm represents an optimization tool that finds its application in the investigations of solar cells, electrical machines, electronic systems etc. [20-23]. The PSO algorithm is established on the population (swarm) of candidate solutions. Also, each particle represents one candidate solution to the problem and moves around in the search spaces by using its experience, as well as the experience of other particles. The movement of each candidate solution (particle) is defined by the speed that is constantly changing in order to find a better feasible solution. Therefore, each particle is flying through *n*-dimensional search space in finding the right position according to the mathematical formulation. The aim of the iterative procedure is to enable the particles to find better positions (Figure 2).

Fig. 2 Flowchart of PSO algorithm

For solving different optimization problems, an objective function *J* should be proposed. In this study, we also define the PSO parameters and variables. The objective function is defined in each particle of search space *A* as follows:

$$
J: A \to R, A \subset R^n
$$

This function shows that for each particle of search space *A* an appropriate value of function *J* is assigned. The value of variable (position of particle with its velocity) is limited due to constraints in the search space that can be found in each iteration. Since we treat the minimization problem, it means that while the value of objective function is lower, than the position of particle is better. In this phase, we have personal best and global best minimum (value and position). It is worth mentioning that the value of global minimum is common for each particle and very close to the minimum of the objective function.

If we define x_i to be position vector and v_i to be velocity vector of particle *i*, P_i represents its best position while *g* is a current global optimum, then moving among the search space can be formulated as

$$
v_i(t+1) = w \cdot v_i(t) + c_1 r_1 (P_i(t) - x_i(t)) + c_2 r_2 (g(t) - x_i(t))
$$
\n(8)

$$
x_i(t+1) = x_i(t) + v_i(t+1)
$$
\n(9)

where r_1 and r_2 are arbitrary positive numbers between 0 and 1, c_1 and c_2 represent accelerate constants, *w* is an inertia weight coefficient and *t* is a current iteration number. Accelerate constants *c¹* and *c²* have a huge impact on the convergent speed because in the case that the values of constants are small, the particle swarm slowly converges to the solution. Otherwise, in a situation where the values of constants are relatively high, the whole optimization process may become unstable.

In this study, we estimated the values of four parameters while the objective function is mean square error of the reference signal and estimated signal value. Therefore, the ISE (*Integral Square Error*) function has been selected as the optimum function, with its mathematic form:

$$
OF1 = \int_0^\infty e^2(t)dt\tag{10}
$$

where $t -$ is time, and $e(t) -$ is the difference between the desired ship heading angle (ψ_d) and actual ship heading angle (ψ) . In this paper we determined the PID parameters for step change of desired ship heading angle of ψ_d =10 [deg].

However, beside the proposed optimum function used in [17], here we propose the following objective function:

$$
OF2 = \int_0^\infty e^2(t)dt + G \cdot max(y)
$$
\n(11)

that deals with the maximal value of rise. So, the objective and priority are to minimize the value of rise. In this paper, the value of coefficient *G* is set to be 10000.

The constraints of the used optimization technique in the paper are $(k_c, T_c, T_c'$ and T_I) which must be set within some pre-specified limits. These limits may be bounded by

$$
k_c^{min} \le k_c \le k_c^{max}, T_c^{min} \le T_c \le T_c^{max}
$$
\n
$$
(12)
$$

$$
T_c^{\prime \min} \le T_c^{\prime} \le T_c^{\prime \max}, T_1^{\min} \le T_1 \le T_1^{\max} \tag{13}
$$

where the superscripts *min* and *max* speak for the minimum and the maximum values of the respective variables.

The procedure of determining the optimal values of PID controller parameters is described as follows. In its basis, PSO algorithm generates the values of PID controller parameters. Upon that, the value of objective function is reported for the input signal. In the next step (iteration), PSO algorithm generates new values of PID controller parameters with the new value of objective function. In the case that this objective function value is lesser, the algorithm is taking the corresponding PID controller parameter values. The procedure is repeated until the best value of objective function is obtained or upon the total number of reported iterations is finished.

However, during the optimization process, we tested the value of rudder angle (δ) . Namely, if its value is greater than 35, the obtained combination of parameters is rejected. In that manner the contribution of the methodology provided in this paper differs from those proposed in [17] which gives an added value in the research area.

4. Simulation results

The proposed method (PSO algorithm, with limitation of rudder angle δ , together with novel optimization function) is used for PID parameters determination of a ship model whose parameters are as follows: k_p =-0.0834, T_p =5.98 [15]. In addition, it is assumed that the natural frequency ω_0 is 0.1 rad/s while damping coefficient ξ is 0.9 [15].

The estimated PID parameters values, determined by using several methods, are presented in Table I. Namely, it shows the results obtained by using Nicolau [15], Calasan [17], as well as by using the proposed method based on the use of PSO algorithm together with objective functions OF1 and OF2.

Parameters	Nicolau $[15]$	Calasan [17]	Proposed $method - OF1$	Proposed $method - OF2$	
K.	-1.2	-1.1102	-3.2715	-1.5606	
T_c	28	30.8	71.9739	40.1966	
	5.98	6.4569	6.5431	18.7858	
	3.57	3.3035	6.1581	8.3948	

Table 1 Comparison of results in terms of parameters value

The step responses of the closed-loop transfer function for all four cases are illustrated in Figure 3a. The corresponding the rudder angle responses are presented in Figure 3b.

a)

Fig. 3 a) Step response of the closed-loop system, **b)** Corresponding rudder angle responses.

The ramp responses of the proposed system are illustrated in Figure 4a. The corresponding the rudder angle responses are presented in Figure 4b.

Fig. 4 a) Ramp response of the closed-loop system, **b)** Corresponding rudder angle responses.

As it can be seen, in both cases (step and ramp input signal), PID autopilot assures null stationary error for both step and ramp variations on reference inputs. However, it can be seen that the OF2 enables obtaining much better signal in terms of overshoot. Also, it can be seen that in all analyzed cases the maximal value of rudder angle is less than the prescribed value. Likewise, the higher change of the desired course angle value leads to higher changes of rudder angle.

In addition to the above mentioned, this paper gives a comparison of the results of the system's step response with the effect of the disturbance and with some results obtained by applying other methods known from the literature. The results are shown in Table 2. As can be seen, the *Calasan* [17] method and the proposed OF1 method have the largest settling time. However, the delay time, the maximum time and the rise time are the least for the proposed methods OF1 and OF2. Moreover, the overshoot value in case of using OF2 is less than the overshoot value obtained when using the *Lyapunov* [19] or *Nicolau* [15] methods. Therefore, since the response rate is important for the response quality (low rise time value and low overshoot time value), with the lowest possible overshoot value, it is clear that the proposed method based on OF2 has very good performance. Although the overshoot value that occurs in the case of using the parameters obtained with the proposed OF2 is greater than zero (i.e. greater than the ESO - Extended state observer [18] and IOL - Input-Output Linearization [18] methods), it should be noted that the rise time of the IOL and ESO methods is significantly higher.

Parameters	Overshoot $\lceil\% \rceil$	Time of overshoot [s]	Delay time [s]	Settling time $[s]$	Rise time [s]
ESO [18]			12.82	> 50	> 25
Lyapunov ^[19]	22	25	8.7	> 60	>10
IOL[18]			13	>49	> 25
Nicolau [15]	27.3	29.5	8.37	> 65	11
Calasan [17]	23	32.5	8.43	74	11.9
Proposed method – simple OF1	35	16.4	5.8	75	7.3
Proposed method – novel OF2		16.1	5.6	54	6.4

Table 2 Characteristics of obtained signals

Maja Krčum, Nikola Marvučić

It is important to point out that in case when steering machine limitations (rudder angle saturation) is not applied, the following results will be obtained: K_c =-85.8305, T_c =20.6428, T_c =7.6052 and T_1 =0. Although the results guarantee the ideal response – almost minimal overshoot with a very short time-delay, they are not realistic, since the complex ship system dynamics cannot follow the change of the input – control signal with adequate speed.

4.1 Robustness analysis

The observed real system contains different kinds of uncertainties and various disturbances due to its complexity. For that reason, in this paper, the robustness analysis of the observed system with tuned parameters was carried out in three ways – by step changing of reference signal (desired ship heading angle), by combined step and ramp changing of reference signal and by adding certain disturbance signal on the output side of the diagram (on ship heading angle – see Figure 1).

In the first case, experiential values of step changes of input signal (desired ship heading angle) were made several times (Figure 5). First, the reference value of yaw angle was reduced by 30%, then after a certain time this value was increased by 50%, and finally the current value was decreased by 20%. As can be seen, for all given changes of reference values, both positive and negative, the best responses were provided by the proposed method OF2. It is clear that the speed of establishing a new stationary state is the highest, while the overshoot value for all the step changes of the input signal is the smallest. Also, the values of rudder angle changes are in permitted limits (see Figure 5b).

In the second case (Figure 6) the referent value of desired course is firstly changed with step signal and after that with ramp signal (see Figure 6a). Just like in the first case the best responses were provided by the proposed method OF2, while in all cases the rudder angle is within the permitted limits (see Figure 6b).

In the case of testing the effect of the step disturbance at the measured value of the actual course angle, the corresponding results are shown in Figure 7. In this case, the step disturbances were added at the output signal (see Fig. $1 -$ disturbance signal). As can be seen, the system closely follows all the changes at the output. It can be noticed that the fastest system response is achieved when the parameters used were determined by the proposed method OF2. Moreover, after the disturbance, the system quickly returns to the stationary state (see Figure 7a and 7b). Note, at starting time the step change of desired course angle is also realized.

Fig. 5 a) Heading course responses on step changes of desired course angle value, **b)** Corresponding rudder angle responses.

Fig. 6 a) System response on combined step and ramp changes of desired course, **b)** Corresponding rudder angle responses.

Therefore, based on all of the above, it can be concluded that the proposed method for PID parameters design enables very secure tracking of the reference signal as well as very secure disturbance attenuation, without an unallowed value of rudder angle.

Fig. 7 a) Additional signal and system responses, **b)** Corresponding rudder angle response

5. Conclusion

This paper deals with the design of the PID controllers for autopilots. For that purpose, a block diagram (of the ship and the autopilot controller) was observed, while the design was realized by using a PSO method.

However, unlike previous works in this field, when designing the parameters of the PID regulator, the importance of taking into account the dynamics (limitations) of the system is emphasized. Namely, by taking the steering dynamics limitations into account, the procedure for selecting the parameters of the PID controller is also defined. In addition to criterion functions for determining the parameters of the controller known from the literature, a new criterion function is proposed which takes into account the maximum overshoot value during the rapid change of the control signal. Moreover, the obtained response results for the step disturbance of the input signal are compared with the responses obtained by using several methods know from the literature. It has been shown that the selected parameters of the controller meet the stability criterion, while providing a fast and efficient system response to the effect of the input step and ram disturbances.

In the future work the authors plan to determine the optimal values of PID parameters for different desired values of ship heading angles. Also, we will test different optimization techniques for this purpose.

REFERENCES

- **[1]** S. Hajizadeh, M. S. Seif, H. Mehdigholi, Evaluation of Planing Craft Maneuverability Using Mathematical Modeling, Brodogradnja: Teorija i praksa brodogradnje i pomorske tehnike Vol. 67, No 1, pp. 85-100, 2016. http://dx.doi.org/10.21278/brod67105
- [2] S. Richards, Electronics, Navigational Aids and Radio Theory for Electrotechnical Officers, Adlard Coles Nautical, Bloomsbury, London-New Delhi-New York-Sydney, 2013.
- [3] F. C. Korkmaz, M. E. Su, F. Alarçin, Control of a Ship Shaft Torsional Vibration Via Modified PID Controller, Brodogradnja: Teorija i praksa brodogradnje i pomorske tehnike Vol. 65, No 1, pp. 17-27, 2014.
- [4] N. Kjerstad, Electronic and Acoustic Navigation Systems (for Maritime Studies), NTNU, Norwegian University of Science and Technology, First edition, Aalesund, Norway, 2016.
- [5] J. Van Amerongen, Adaptive Steering of Ships: A Model Reference Approach, Automatica, vol. 20, no. 1, pp. 3–14, 1984. [https://doi.org/10.1016/0005-1098\(84\)90060-8](https://doi.org/10.1016/0005-1098(84)90060-8)
- [6] T. Lauvdal, T. I. Fossen, Robust Adaptive Ship Autopilot With Wave Filter and Integral Action, Int. J. Adapt. Control Signal Process, vol. 12, pp. 605-622, 1998. [https://doi.org/10.1002/\(SICI\)1099-](https://doi.org/10.1002/(SICI)1099-1115(199812)12:8%3C605::AID-ACS516%3E3.0.CO;2-1) [1115\(199812\)12:8<605::AID-ACS516>3.0.CO;2-1](https://doi.org/10.1002/(SICI)1099-1115(199812)12:8%3C605::AID-ACS516%3E3.0.CO;2-1)
- [7] T. A. Johansen, T. P. Fuglseth, P. Tondel, and T. I. Fossen, Optimal Constrained Control Allocation in Marine Surface, J. Control Eng. Practice, vol. 16, pp. 457–464, 2008. <https://doi.org/10.1016/j.conengprac.2007.01.012>
- [8] S.D. Lee, C.Y. Tzeng, W.W. Huang, Ship Steering Autopilot Based on Anfis Framework and Conditional Tuning Scheme, J. Maritime Eng. Front., vol. 1, no. 3, pp. 53–62, Aug. 2013.
- [9] R. S. Burns, The Use of Artificial Neural Network for the Intelligent Optimal Control of Surface Ships, IEEE J. Ocean. Eng., vol. 20, no. 1, pp. 65–72, Jan. 1995. DOI: <https://doi.org/10.1109/48.380245>
- [10] S. L. Dai, C. Wang, F. Luo, Identification and Learning Control of Ocean Surface Ship Using Neural Networks, IEEE Trans. Ind. Inf., vol. 8, no. 4, pp. 801–810, Nov. 2012. DOI: <https://doi.org/10.1109/TII.2012.2205584>
- [11] G. Rigatos and S. Tzafestas, Adaptive Fuzzy Control for the Ship Steering Problem, J. Mechatron., vol. 16, pp. 479–489, 2006. <https://doi.org/10.1016/j.mechatronics.2006.01.003>
- [12] B. Samanta and C. Nataraj, Design of Intelligent Ship Autopilots using Particle Swarm Optimization, 2008 IEEE Swarm Intelligence Symposium, St. Louis MO USA, September 21-23, 2008 <https://doi.org/10.1109/SIS.2008.4668327>
- [13] T. I. Fossen and J. P. Strand, Tutorial on nonlinear backstepping: Application to ship control, Model. Identif. Control, vol. 20, no. 2, pp. 83–134, Mar. 1999. [DOI:](http://www.doi.org/)<https://doi.org/10.4173/mic.1999.2.3>
- [14] C. C. Lim and W. Forsythe, "Autopilot for ship control. Part 1: Theoretical design," IEE Proceedings D Control Theory and Applications, vol. 130, no. 6, pp. 281-287, November 1983. <https://doi.org/10.1049/ip-d.1983.0048>
- [15] V. Nicolau, On PID Controller Design by Combining Pole Placement Technique with Symmetrical Optimum Criterion, Mathematical Problems in Engineering, vol. 2013, Article ID 316827, 8 pages, 2013. <https://doi.org/10.1155/2013/316827>
- [16] A. A. Godbole, T. R. Libin, S. E. Talole, Extended State Observerbased Robust Pitch Autopilot Design for Tactical Missiles, Proc. Inst. Mech. Eng. G, J. Aerosp. Eng., vol. 226, no. 12, pp. 1482–1501, Dec. 2011. <https://doi.org/10.1177/0954410011426397>
- [17] M. Calasan, T. Dlabac, N. Marvucic, PID autopilot design for heading control problem of a conventional ship, International Conference on Transport Science ICTS 2018, Portoroz, 14-15. June 2018, Slovenia, pp. 65-68.
- [18] S. Das and S. E. Talole, Robust Steering Autopilot Design for Marine Surface Vessels, IEEE Journal of Oceanic Engineering, vol. 41, no. 4, pp. 913-922, Oct. 2016. DOI: <https://doi.org/10.1109/JOE.2016.2518256>
- [19] T. I. Fossen, Guidance and Control of Ocean Vehicle, Ch. 3, New York, NY, USA: Wiley, 1994. ISBN 0471941131.
- [20] H. Shi, H. Wen, Y. Hu and L. Jiang, "Reactive Power Minimization in Bidirectional DC–DC Converters Using a Unified-Phasor-Based Particle Swarm Optimization," IEEE Transactions on Power Electronics, vol. 33, no. 12, pp. 10990-11006, Dec. 2018. <https://doi.org/10.1109/TPEL.2018.2811711>
- [21] T. S. Babu, J. P. Ram, T. Dragičević, M. Miyatake, F. Blaabjerg and N. Rajasekar, "Particle Swarm Optimization Based Solar PV Array Reconfiguration of the Maximum Power Extraction Under Partial Shading Conditions," IEEE Transactions on Sustainable Energy, vol. 9, no. 1, pp. 74-85, Jan. 2018. <https://doi.org/10.1109/TSTE.2017.2714905>
- [22] M. G. Bijan and P. Pillay, "Efficiency Estimation of the Induction Machine by Particle Swarm Optimization Using Rapid Test Data With Range Constraints," IEEE Transactions on Industrial Electronics, vol. 66, no. 8, pp. 5883-5894, Aug. 2019. <https://doi.org/10.1109/TIE.2018.2873121>
- [23] A. Safari, H. Shahsavari and F. Babaei, "Optimal Design of Controllers for Power Network Connected SOFC Using of Multi-objective PSO", Serbian Journal of Electrical Engineering, vol. 15, no.2, 145-163, June 2018. DOI:<https://doi.org/10.2298/SJEE170822001S>
- [24] SOLAS Consolidated Edition 2014, IMO, London, 2014.

