

# Selection of PID controller design plane for time-delay systems using genetic algorithm

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**Abstract** – The design of a Proportional-Integral-Derivative (PID) controller with proportional, integral, and derivative, gain,  $k_p$ ,  $k_i$ , and  $k_d$ , respectively, for a time-delay system, is quite common, particularly in the  $k_i$ - $k_d$  plane, for a fixed  $k_p$  or in the  $k_p$ - $k_i$  plane, for a fixed  $k_d$ . These design methods have been widely reported in the literature, however, the process of investigating the effects of using any of these design planes on system performance has not been given serious attention hence the need for this study. The stability region in the  $k_i$ - $k_d$  and  $k_p$ - $k_i$  design plane for a fixed value of  $k_p$  and  $k_d$  respectively were determined. For every determined stability region, the optimum value of controller gains in the plane was determined using a genetic algorithm (GA) with the integral of time multiplied by absolute error (ITAE) used as the objective function. The optimum value of the fixed gains was graphically determined by plotting the minimum of ITAE (Min-ITAE) for each stability region against the fixed gains. The overall optimum controller gains are the fixed gain that gives minimum of Min-ITAE (Min (Min-ITAE)) and the gains that resulted in Min-ITAE that yielded the Min (Min-ITAE). Using the determined overall optimum controller gains, the system closed-loop step response was plotted for the two design planes and the time domain performance measures (TDPMs) were determined. Based on TDPMs obtained for examples 1, 2, and 3, the  $k_i$ - $k_d$  design plane yielded a faster response while the  $k_p$ - $k_i$  design plane yielded a response that closely tracks the input irrespective of the system type and order. The study will enable control system designers to select the design plane that will give the best system performance right from the start of controller design without involving trial and error once the system transfer function and design specifications are known.

**Keywords:** Genetic algorithm, ITAE, PID controller, Stability region, Time-delay system, Time domain performance measures

## 1. INTRODUCTION

All practical control systems have associated time delays [1-3] resulting from the processing and transmission of signals in the control loop [4, 5]. Therefore, the designing of a controller for such a system requires a full understanding of the effects of this delay on the system's performance. Time delay can degrade the system's quality of performance or destabilise the system in the worst case [6]. To control this class of systems, a PID controller is normally employed due to its popularity [7], simplicity, robustness, and ease of use [8-10]. To design a PID controller for a time-delay system the first step is to establish the stability boundary in the space of the controller parameter [11]. This is because of the

lots of work that have been done in this area to the most recent, designs of PID controller for time-delay systems are normally carried out using the stability locus method [12].

One of the proven steps for the design of a PID controller for time-delay systems using the stability locus method involves plotting the stability boundaries in the  $k_i$ - $k_d$  plane for a fixed  $k_p$  or in the  $k_p$ - $k_i$  plane for a fixed  $k_d$  and then computing all the stabilizing values of  $k_p$ ,  $k_i$ , and  $k_d$  in the stability region by sweeping over  $k_p$  and  $k_d$  for  $k_i$ - $k_d$  and  $k_p$ - $k_i$  design planes respectively [16, 17].

The method for computation of all the stabilizing PID controller gains for a linear arbitrary order system with time delay in the  $k_i$ - $k_d$  plane with fixed  $k_p$  was re-

ported in [7], while the procedure for computing the entire stability gains in the  $k_p$ - $k_i$  plane with fixed  $k_d$  was explained in [13]. The stabilization of controller parameters in the  $k_p$ - $k_i$  plane for a given  $k_d$  for a time-delay integral fractional-order system, under the control of a fractional-order PID controller, was earlier determined and reported in [14]. It was observed that the controller gains in the  $k_p$ - $k_i$  plane for a given  $k_d$  and other controller parameters yielded a general stability region, while the global stability region was obtained after the gridding of  $k_d$  PID parameters for a second-order plant with time delay were obtained in the  $k_p$ - $k_i$  plane with a fixed  $k_d$  for specified GM and PM using the dominant poles method, as reported in [13]. PID controller tuning method for integrating system with time delay was developed in the  $k_p$ - $k_i$  plane for a fixed  $k_d$  by [15]. After determining the stability region, weighted geometrical center approach was used to determine the required PID controller gains. The results obtained via simulation are good, but the fixed  $k_d$  was not optimized. In [16, 17], parametric methods were used to establish the stability region in the  $k_i$ - $k_d$  plane for a fixed  $k_p$  and  $k_p$ - $k_i$  plane for a fixed  $k_d$ , and GA was used to determine the optimum controller gains in the established region of the plane.

Despite the huge amount of published works in the literature on PID controller designs for time-delay systems in either  $k_i$ - $k_d$  or  $k_p$ - $k_i$  design planes, the effects of using either of the two design planes on the performance of the designed system have not been given serious attention in the research community, hence the need for this study. The aim of this study, therefore, is to investigate the effects of PID controller design plane selection on the performance of the designed system.

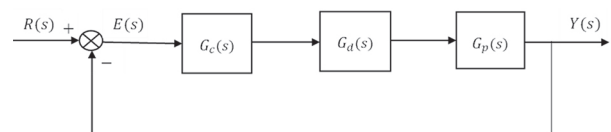
To achieve the aim of this study, the equations that relate the controller gains, the system parameters, and the time delay together were derived. Using these equations and the frequency obtained from the graph of  $k_p$  and  $k_d$  against the frequency ( $\omega$ ), the stability boundaries in  $k_i$ - $k_d$  and  $k_p$ - $k_i$  planes for fixed  $k_p$  and  $k_d$  respectively was plotted. After the determination of the stability boundaries, the optimum controller gains within the stability region can now be determined. Several optimization methods have been used in the literature but in this study, GA was used because of its heuristic characteristics, powerful searching capabilities [18], and amenability of the method of stability boundary locus (used in this study) to GA. For each of these regions, the optimum controller gains were determined via GA using ITAE as the objective function. Since GA was used as a minimization process in this study, the obtained minimum value of ITAE (Min-ITAE) was plotted against the fixed  $k_p$  and  $k_d$  for  $k_i$ - $k_d$  and  $k_p$ - $k_i$  design planes respectively. Based on these graphs, the overall optimum gains were determined. These gains were used together with the system and time-delay transfer functions to generate the system closed-loop unit step response, which yielded the system TDPMs.

The TDPMs considered in this study are  $T_r$  (rise time), %OS (percentage overshoot), %US (percentage undershoot),  $T_p$  (peak time),  $T_s$  (settling time), and  $ess$  (steady-state error). These performance measures were used for system analysis and characterisation in the plane under consideration. The results show that the  $k_i$ - $k_d$  design plane produced a faster response while the  $k_p$ - $k_i$  design plane gives a response that closely tracks the input irrespective of the system's type and order. Therefore, the major contribution of the study is to aid the control system designers in the selection of the design plane that will give the best system performance at the beginning of controller design without involving trial and error, given that the system transfer function and design specifications are known.

## 2. METHODOLOGY

### 2.1. DETERMINATION OF STABILITY BOUNDARY FOR PID-CONTROLLED SYSTEM

The derivation of system forward and closed-loop transfer functions used in this study is based on the block diagram of the unity feedback control system shown in Fig.1, where  $R(s)$ ,  $E(s)$ , and  $Y(s)$  are the reference input, error, and output respectively. Equations (1), (2), and (3) are the adopted expression for the plant, time delay, and controller transfer function respectively.



**Fig. 1.** Block diagram of unity feedback time-delay control system

$$G_p(s) = \frac{N(s)}{D(s)} \quad (1)$$

where  $N(s)$  and  $D(s)$  are the plant transfer function numerator and denominator respectively.

$$G_d(s) = e^{-\tau s} \quad (2)$$

where  $\tau$  is the time delay in sec.

$$G_c(s) = \frac{k_d s^2 + k_p s + k_i}{s} \quad (3)$$

For easy application of D-decomposition method, the numerator and denominator of Equation (1) were broken into their even and odd parts after substituting  $j\omega$  for  $s$  as shown in Equation (4). It should be noted that for compactness purpose, the  $(-\omega^2)$  term has been removed from  $N_e(-\omega^2)$ ,  $N_o(-\omega^2)$ ,  $D_e(-\omega^2)$  and  $D_o(-\omega^2)$  in Equation (4) [19].

$$G_p(j\omega) = \frac{N_e + j\omega N_o}{D_e + j\omega D_o} \quad (4)$$

Finding the stability boundary in  $k_p$ ,  $k_i$  and  $k_d$  space is a three-dimensional problem. For easy controller de-

sign and analysis, it can be reduced to a two-dimensional problem by fixing one of the parameters and finding the stability region in the plane of the remaining two parameters. The overall stability region can be determined using the stability regions in the plane of the two parameters by sweeping over the fixed parameter values. For this study,  $k_p$  and  $k_d$  were fixed and the stability boundaries in the  $k_i$ - $k_d$  and  $k_p$ - $k_i$  planes were determined respectively.

The equations and conditions required for determining the stability region in the  $k_i$ - $k_d$  plane for a fixed  $k_p$  and the stability region in the  $k_p$ - $k_i$  plane for a fixed  $k_d$  are presented as follows [16, 17, 19, 20]:

### 2.1.1. Stability region in the $k_i$ - $k_d$ plane for a fixed $k_p$

For  $\omega=0$

$$k_i=0 \quad (5)$$

For  $\omega>0$

$$k_p = \frac{(\omega^2 N_o D_o + N_e D_e) \cos(\omega T) + \omega (N_o D_e - N_e D_o) \sin(\omega T)}{-(N_e^2 + \omega^2 N_o^2)} \quad (6)$$

$$k_d = \frac{\omega^2 (N_o D_e - N_e D_o) \cos(\omega T) - \omega (N_e D_e + \omega^2 N_o D_o) \sin(\omega T) + k_i (N_e^2 + \omega^2 N_o^2)}{\omega^2 (N_e^2 + \omega^2 N_o^2)} \quad (7)$$

where  $\omega = \omega_n$ , ( $n=1,2,\dots$ ) are the frequencies at which the line of a given value of  $k_p$  intercepts the graph of  $k_p$  of Equation (6) versus  $\omega$ , and  $n$  is the number of points of intersection or the number of lines obtainable from Equation (7).

The stability boundary in the  $k_i$ - $k_d$  plane is formed by the line obtained from Equation (5) and the lines generated from Equation (7) when  $\omega_n$  is substituted for  $\omega$ .

### 2.1.2. Stability region in the $k_p$ - $k_i$ plane for a fixed $k_d$

For  $\omega=0$

$$k_i=0 \quad (8)$$

For  $\omega \geq 0$

$$k_p = \frac{(\omega^2 N_o D_o + N_e D_e) \cos(\omega T) + \omega (N_o D_e - N_e D_o) \sin(\omega T)}{-(N_e^2 + \omega^2 N_o^2)} \quad (9)$$

$$k_i = \frac{\omega^2 (N_o D_e - N_e D_o) \cos(\omega T) - \omega (N_e D_e + \omega^2 N_o D_o) \sin(\omega T) - K_d \omega^2 (N_e^2 + \omega^2 N_o^2)}{-(N_e^2 + \omega^2 N_o^2)} \quad (10)$$

where  $\omega=[0,\omega_c]$  and  $\omega_c$  is the frequency at which the line of a given value of  $k_d$  in Equation (10) intercepts the graph of  $k_d$  in Equation (11) against  $\omega$ . Equation (11) was obtained from equation (7) when it is assumed that  $k_i=0$ .

$$k_d = \frac{\omega^2 (N_o D_e - N_e D_o) \cos(\omega T) - \omega (N_e D_e + \omega^2 N_o D_o) \sin(\omega T)}{\omega^2 (N_e^2 + \omega^2 N_o^2)} \quad (11)$$

The stability boundary in the  $k_p$ - $k_i$  plane is formed by the line obtained from Equation (8) and the locus generated by Equations (9) and (10) when  $\omega=[0,\omega_c]$ .

## 2.2. OPTIMIZATION OF PID CONTROLLER PARAMETERS

After the determination of the system convex stability region, the next step is to determine the value of the

controller parameters that give the best system performance. The success of the determination and selection of optimum PID controller parameters using GA as an optimization tool is strongly dependent on the selection of appropriate objective function. To optimize the PID controller parameters, different system performance indices have been used as the objective function. ITAE, ISE, MSE, and IAE were used by [21], ISE, ITAE, and IAE were used in [22, 23], while in [24] work, ISE, ITAE, IAE, and their combination were used. The ITAE (see equation (12)) was adopted in this study due to its best selectivity, a good criterion for PID controllers design [25]; and its minimum value can be easily defined as the system parameters are varied [26],

$$ITAE = \int_0^{\infty} t|e(t)| dt \quad (12)$$

where  $T_1 \geq T_s$ , ( $T_s$  is the system settling time in sec.),  $e(t)$  is the error and  $t$  is the time in sec.

## 2.3. GA-BASED PID CONTROLLER PARAMETERS OPTIMIZATION

The PID controller gains were optimised using GA by following the steps shown in the flowchart of Fig. 2.

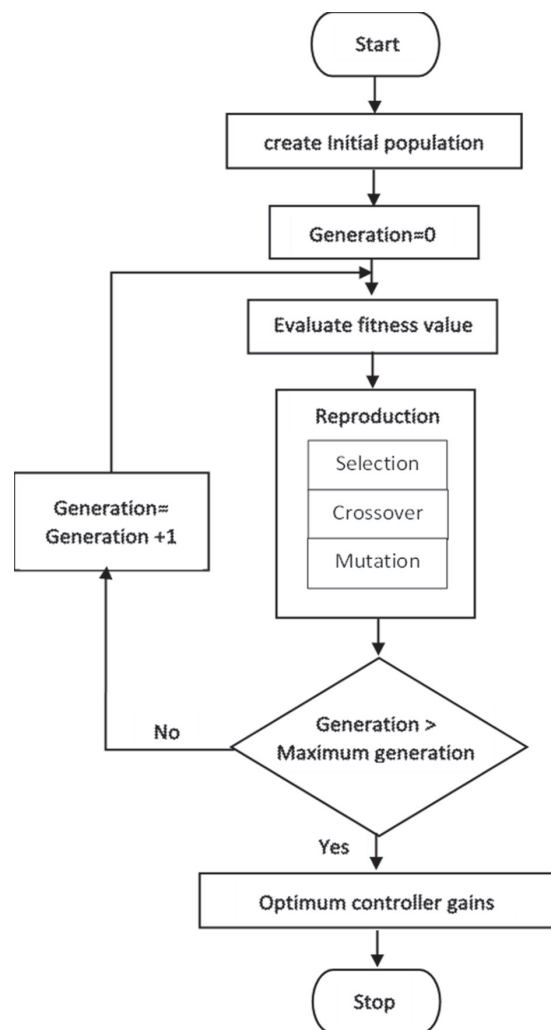


Fig. 2. Flowchart of GA process for optimising PID controller gains [27, 28]

The initial population was created inside the stability region because the stability boundary serves as the constraint for the optimisation problem. To determine the fitness of the generated PID controller gains, the objective function, ITAE in this case was calculated for these gains in parallel. Based on the ranking value of the fitness function for the PID controller gains, half of the population was selected for reproduction. To get the next generation, the crossover was carried out on the already selected feasible solutions (gains). Also, to guide against the production of false optimum controller gains, a few individual solutions in the generation were mutated. The process continues in a loop until when there are no significant changes in the generations which is the condition for termination of the optimisation process. Details on the design of PID controllers for delay-free and time-delay systems using GA can be found in [16, 17] and [29] respectively.

### 3. SIMULATION EXAMPLES

Three different examples were used to demonstrate the selection of the PID controller design plane. For each of these examples, GA was used in searching for the: (i) optimum  $k_i$  and  $k_d$  in the  $k_i - k_d$  plane for different values of  $k_p$  (ii) optimum  $k_p$  and  $k_i$  in the  $k_p - k_i$  plane for different values of  $k_d$ . Associated with each set of the optimum gains is a unique Min-ITAE since GA was used as a minimisation process. The optimum fixed  $k_p$  gain  $k_{p,f, opt}$  for the  $k_i - k_d$  plane and  $k_{d,f, opt}$  for the  $k_p - k_i$  plane is the one that gives Min(Min-ITAE) and can be graphically determined from the graphs of Min-ITAE versus fixed  $k_p$  and Min-ITAE versus fixed  $k_d$  for  $k_i - k_d$  plane and  $k_p - k_i$  plane respectively. The overall optimum gains for the  $k_i - k_d$  plane are the  $k_{p,f, opt}$  and the optimum  $k_i$  and  $k_d$  denoted by  $k_{i, opt}$  and  $k_{d, opt}$  respectively, that yielded the Min-ITAE that gives Min(Min-ITAE). Also, the overall optimum gains for the  $k_p - k_i$  plane are the  $k_{d,f, opt}$  and the optimum  $k_p$  and  $k_i$  also denoted by  $k_{p, opt}$  and  $k_{i, opt}$  respectively, that yielded the Min-ITAE that gives Min(Min-ITAE). To provide room for system performance analysis and characterisation in each of these design planes using TDPMS, the system step response was plotted based on the overall optimum controller gains.

#### 3.1. EXAMPLE 1

The design of the PID controller for an integrating second-order time-delay system with the system transfer function given by equation (13) [30] was considered in this example. For this system  $N_e=1$ ,  $N_o=0$ ,  $D_e=-\omega^2$ , and  $D_o=1$ .

$$G(s) = \frac{1}{s(s+1)} e^{-s} \quad (13)$$

The ranges of  $k_d$  and  $k_p$  were first determined by plotting  $k_d$  against  $k_p$  as shown in Fig. 3 using equations (6) and (11). Also based on these equations, the plots of  $k_p$  against  $\omega$  and  $k_d$  against  $\omega$  are shown in Fig. 4 and 5 for the determination of the relevant frequencies for any given value of  $k_p$  and  $k_d$  respectively. It can be seen from Fig. 3 that the ranges of  $k_p$  and  $k_d$  are 0 – 1.717 and -1 – 2.261 respectively.

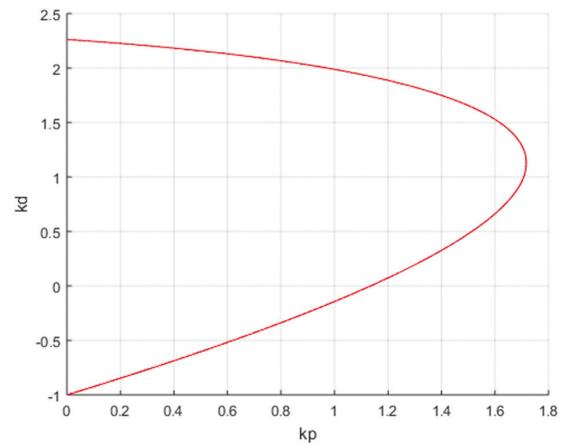


Fig. 3. Plot of  $k_d$  against  $k_p$

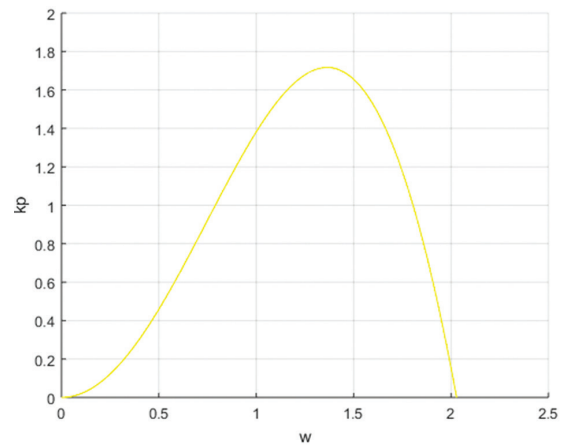


Fig. 4. Plot of  $k_p$  against  $\omega$

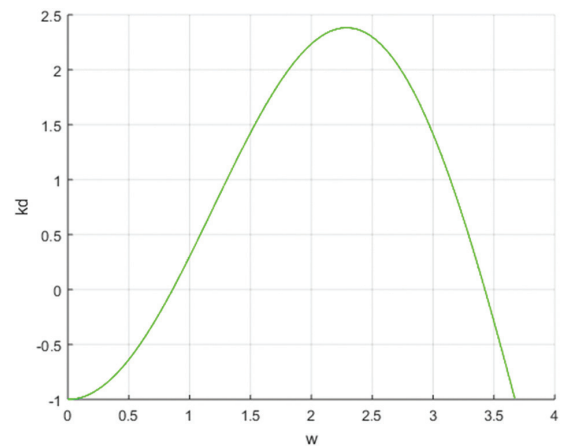


Fig. 5. Plot of  $k_d$  against  $\omega$

For the case of PID controller design in the  $k_i - k_d$  plane, the fixed gains considered based on the range of  $k_p$  as presented in Fig. 4 are  $k_p = 0.2, 0.4, 0.6, 0.8, 1.0, 1.2, 1.4$ , and 1.6. The detailed design for  $k_p = 0.2$  is presented as follows:

As explained in Subsection 2.1.1., when  $\omega = 0$ , the stability boundary equation that can be used for generating one of the boundary lines was derived using Equation (5). To generate the equations that can be used to form the

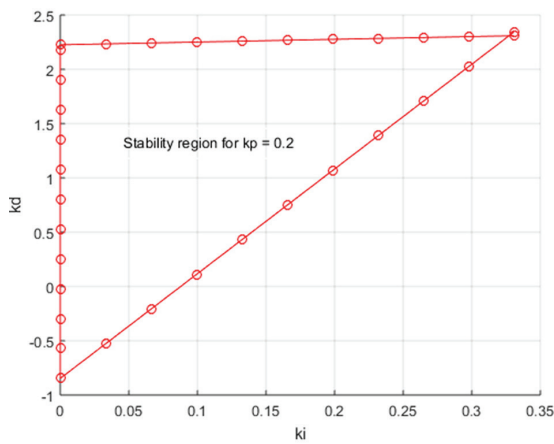
remaining boundary lines, the frequency at the point of intersection of the line of  $k_p = 0.2$  with the plot of  $k_p$  versus  $\omega$  in Fig. 4 was determined. From Fig. 4, the points of intersection are two and their corresponding frequencies are  $\omega_1=0.322$  rad and  $\omega_2 = 1.991$  rad. Substituting  $\omega_1$  and  $\omega_2$  in Equation (7) yielded the remaining two equations required for generating the stability boundary's lines. The resulting stability boundary equations are:

$$k_i=0 \quad (14)$$

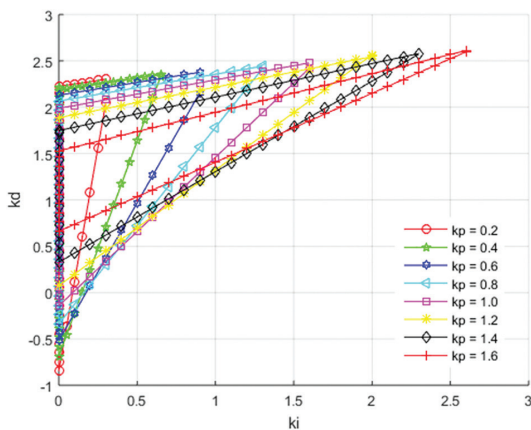
$$k_{d1}=9.6447k_i-0.8467 \quad (15)$$

$$k_{d2}=0.2523k_i+2.2257 \quad (16)$$

Based on Equations (14) – (16), and the similar equations obtained for values of remaining fixed  $k_p$  following, the steps used for  $k_p = 0.2$ , the system stability boundary in the  $k_i$ - $k_d$  plane shown in Fig. 6 and 7 were obtained for when  $k_p = 0.2$  and for the complete set of fixed  $k_p$ , respectively.



**Fig. 6.** The stability boundary in the  $k_i$ - $k_d$  plane for  $k_p = 0.2$



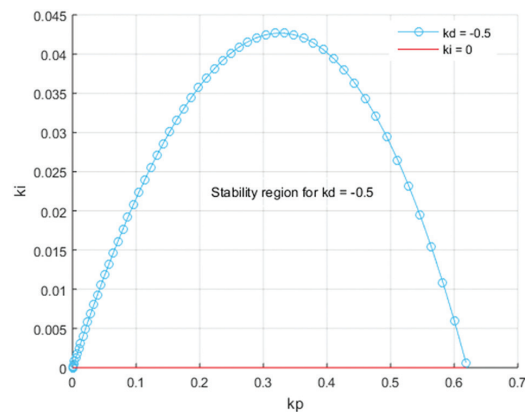
**Fig. 7.** The stability boundary in the  $k_i$ - $k_d$  plane for the set of fixed  $k_p$

The constraints needed for the GA were formed using Equations (14) – (16) and the stability boundary equations for other fixed  $k_p$  considered following the same steps. After the formation of the constraints, the GA MATLAB code was written according to the flowchart in Fig. 2. The code was run for each scenario to

determine the optimum  $k_i$  and  $k_{d,r}$  and the corresponding Min-ITAE for each of the fixed  $k_p$ . The  $k_{p,f,opt}$  and Min (Min-ITAE) were obtained from the plot of Min-ITAE versus fixed  $k_p$ . The required TDPMs were obtained from the unit step response plotted using the determined overall optimum controller gains.

Taking the case of PID controller design in the  $k_p$ - $k_i$  plane the fixed gains considered based on the range of  $k_d$  as shown in Fig. 5 are  $k_d = -0.5, 0, 0.5, 1.0,$  and  $1.5$ . The design analysis for  $k_d = -0.5$  is presented as follows:

As discussed in Subsection 2.1.2., when  $\omega = 0$ , the stability boundary equation that was used for generating one of the boundaries was derived using Equation (8). The frequency at the first point of intersection of the line of  $k_d = -0.5$  with the plot of  $k_d$  versus  $\omega$  in Fig. 5 was determined to generate the equation that can be used to form the remaining boundary locus. From Fig. 5, the frequency at the said points of intersection,  $\omega_c=0.5917$  rad. By generating the straight line defined by Equation (8) and plotting  $k_i$  against  $k_p$  using Equations (9) and (10) with  $\omega=[0, \omega_c]$  the stability boundary of Fig. 8 was obtained for  $k_d = -0.5$ .



**Fig. 8.** The stability boundary in the  $k_p$ - $k_i$  plane for  $k_d = -0.5$

The same steps used for  $k_d = -0.5$  was adopted for other fixed  $k_d$  scenarios and the stability boundaries of Fig. 9 were obtained. The stability boundary equations for  $k_d = -0.5$  case were obtained from Equation (8) and by curve fitting the stability locus of Fig. 8 and are presented in Equations (17) and (18) respectively.

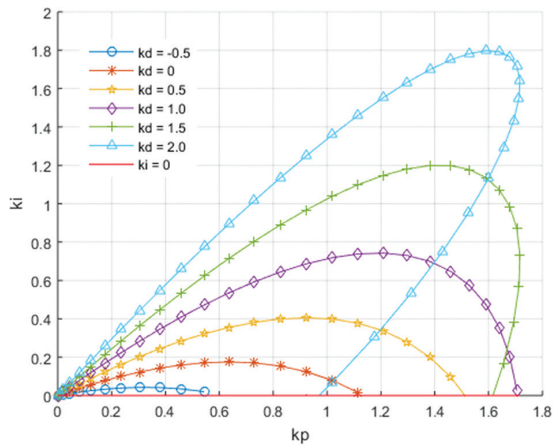
$$k_i=0 \quad (17)$$

$$k_i = -0.061k_p^4 - 0.069k_p^3 - 0.34k_p^2 + 0.25k_p - 3.6 \times 10^{-6} \quad (18)$$

For other fixed  $k_d$  values, the stability boundary equations like (17) and (18) were also generated following the steps used for  $k_d = -0.5$  case.

The constraints needed for the GA were formed using Equations (17) and (18) and the stability boundary equations for other fixed  $k_d$  were considered. After the formation of constraints, the GA MATLAB code was written according to the flowchart in Fig. 2. The code was run for each scenario to determine the optimum  $k_p$  and  $k_i$ ,

and the corresponding Min-ITAE for each of the fixed  $k_d$ ,  $k_{d,opt}$  and Min (Min-ITAE) were obtained from the plot of Min-ITAE versus fixed  $k_d$ . Also, the required TDPMs were obtained from the unit step response plotted using the determined overall optimum controller gains.



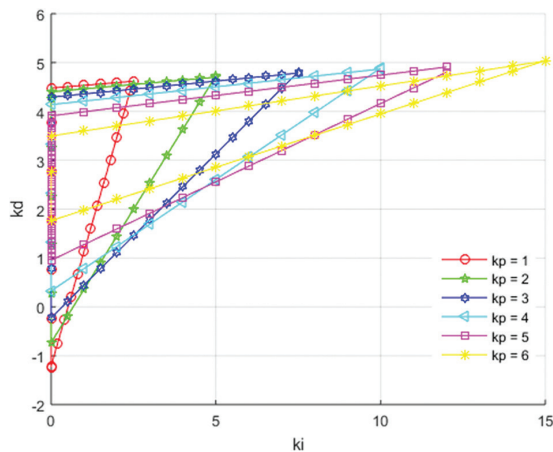
**Fig. 9.** The stability boundary in the  $k_p$ - $k_i$  plane for the set of fixed  $k_d$

### 3.2. EXAMPLE 2

The design of a PID controller for an integrating second-order time-delay system with left-hand side zero whose transfer function is shown in Equation (19) [15] was considered in this example. For this system  $N_e=0.6$ ,  $N_o=-0.18$ ,  $D_e = -w^2$ , and  $D_o=1$

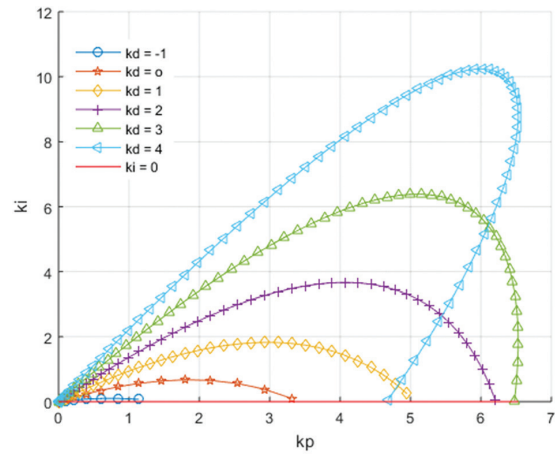
$$G(s) = \frac{0.6(-0.3s+1)e^{-0.2s}}{s(s+1)} \quad (19)$$

The method employed for the design of the PID controller in Example 1 was adopted in this example. As a result, the PID controller design in the  $k_i$ - $k_d$  plane in this example was considered for the following fixed  $k_p$  values: 1, 2, 3, 4, 5, and 6. The stability boundary for the considered values of fixed  $k_p$  is shown in Fig. 10. The required TDPMs were obtained from the unit step response plotted using the determined overall optimum controller gains.



**Fig. 10.** The stability boundary in the  $k_i$ - $k_d$  plane for the set of fixed  $k_p$

To design the PID controller in the  $k_p$ - $k_i$  plane for this example the considered range of fixed  $k_d$  are  $k_d = -1, 0, 1, 2, 3$ , and  $4$ , and the stability boundaries of Fig. 11 were obtained for these fixed values of  $k_d$ . The required TDPMs were obtained from the unit step response plotted using the determined overall optimum controller gains.



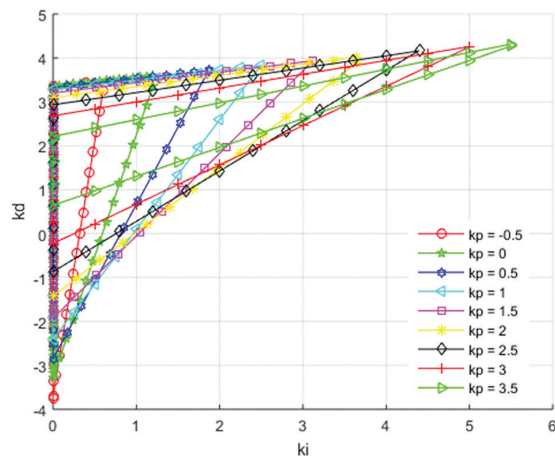
**Fig. 11.** The stability boundary in the  $k_p$ - $k_i$  plane for the set of fixed  $k_d$

### 3.3. EXAMPLE 3

The design of a PID controller for a second-order time-delay system with left-hand side zero whose transfer function is shown in Equation (20) [19] was considered in this example. For this system  $N_e=1$ ,  $N_o=-0.5$ ,  $D_e=1-2w^2$ , and  $D_o=3$ .

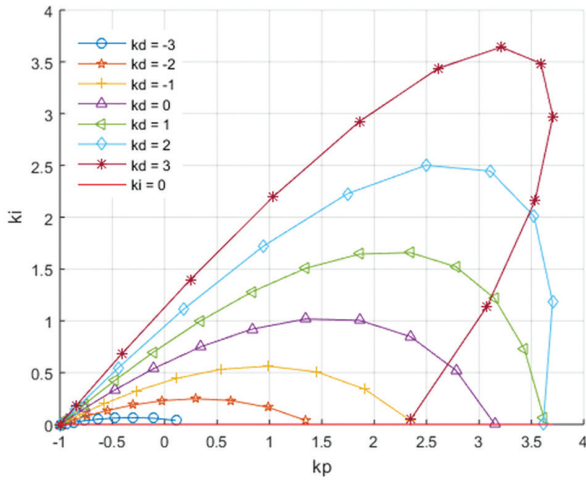
$$G(s) = \frac{(-0.5s+1)e^{-0.6s}}{(s+1)(2s+1)} \quad (20)$$

In this example, the method of PID controller design used in Example 1 was adopted. Therefore, for the  $k_i$ - $k_d$  plane the considered fixed  $k_p$  values are:  $k_p = -0.5, 0, 0.5, 1.0, 1.5, 2.0, 2.5, 3.0$ , and  $3.5$ . For these  $k_p$  values, the obtained stability boundary is shown in Fig. 12. The required TDPMs were obtained from the unit step response plotted using the determined overall optimum controller gains.



**Fig. 12.** The stability boundary in the  $k_i$ - $k_d$  plane for the set of fixed  $k_p$

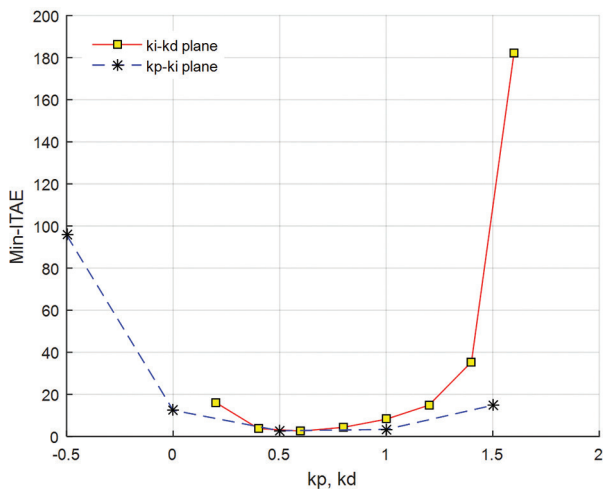
For the design of PID controller in the  $k_p$ - $k_i$  plane the fixed values of  $k_d$  considered are:  $k_d = -3, -2, -1, 0, 1, 2$  and  $3$ . Based on these values of  $k_d$  the stability boundaries plot of Fig. 13 was obtained. The required TDPMs were obtained from the unit step response obtained using the determined overall optimum controller gains.



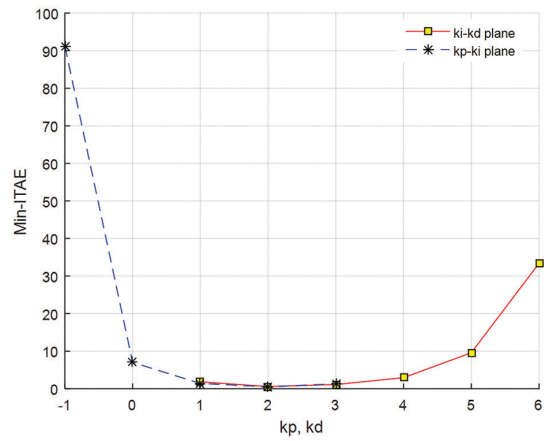
**Fig. 13.** The stability boundary in the  $k_p$ - $k_i$  plane for the set of fixed  $k_d$

#### 4. RESULTS AND DISCUSSIONS

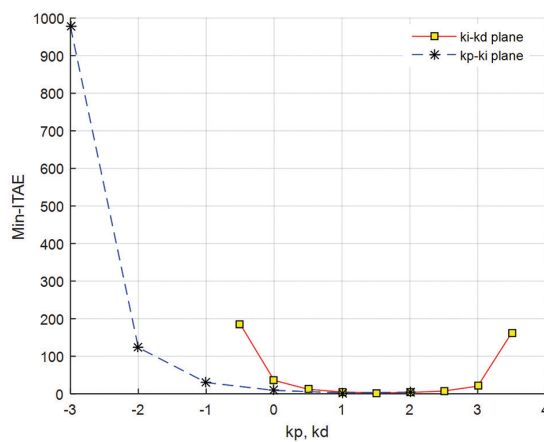
The graphs of Min-ITAE versus the fixed  $k_p$  and fixed  $k_d$  for the  $k_i$ - $k_d$  design plane and  $k_p$ - $k_i$  design plane for Examples 1, 2, and 3 are shown in Fig. 14, 15, and 16 respectively. It can be seen from these Figures that Min-ITAE decreases from the initial value to the minimum value and then increases to a final value. Based on the minimum of Min-ITAE and the corresponding fixed  $k_p$  and fixed  $k_d$  obtained,  $k_{p,opt}$ ,  $k_{d,opt}$  and Min (Min-ITAE) were determined and presented in Table 1. Using the value of  $k_{p,opt}$ ,  $k_{d,opt}$  and the corresponding value of Min (Min-ITAE) the value of  $k_{p,opt}$ ,  $k_{i,opt}$  and  $k_{d,opt}$  were obtained from the GA optimisation results and presented in Table 1.



**Fig. 14.** Plot of Min-ITAE versus the values of fixed  $k_p, k_d$  for Example 1



**Fig. 15.** Plot of Min-ITAE versus the values of fixed  $k_p, k_d$  for Example 2



**Fig. 16.** Plot of Min-ITAE versus the values of fixed  $k_p, k_d$  for Example 3

Using the open-loop transfer functions of Equations (13), (19), and (20) and the overall optimum gains of Table 1, the system closed-loop unit step responses of Fig. 17, 18, and 19 were plotted for Examples 1, 2, and 3 respectively. From Fig. 17, 18, and 19, it is obvious that the systems' response never tracks the unit step input instead, 0.5, 0.375, and 0 were tracked respectively, therefore, the need for a controller for these systems.

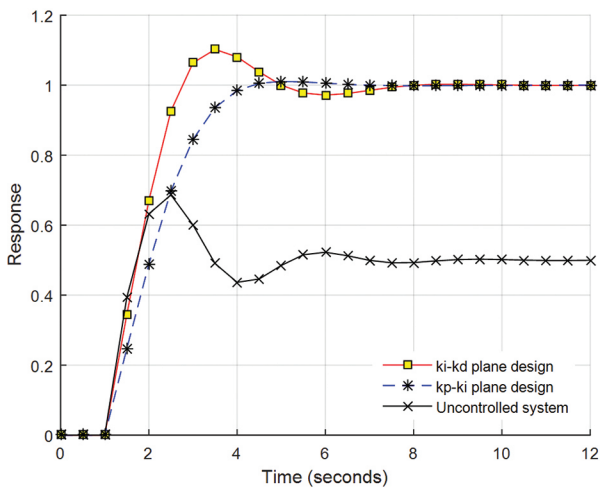
The TDPMs values obtained from the step response of Fig. 17 to 19 for the controlled systems under the two design planes are presented in Table 2. It is obvious from Table 2 that the resulting system from the two design planes for the 3 examples is of good steady state because  $ess=0$ . Considering the transient state status, the response of the  $k_i$ - $k_d$  design plane has higher swiftness for all the examples because of low  $T_r$  and  $T_p$  compared to the  $k_p$ - $k_i$  design plane. On the other hand, the degree of similarity between the response of the system designed in the  $k_p$ - $k_i$  plane with the unit step input for all examples is high compared with that of the  $k_i$ - $k_d$  plane because of the lower value of %OS and  $T_s$  associated with the former. It can also be seen from Table 2, for Examples 2 and 3, there is an associated %US because of the right-hand side zero. But the %US for the  $k_i$ - $k_d$  plane is higher than that of the  $k_p$ - $k_i$  plane.

**Table 1.** Optimum controller gains for different scenarios

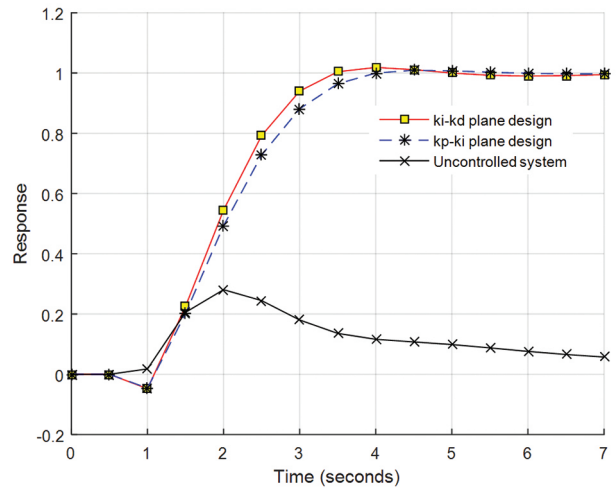
| Example | Plane     | $k_{p,opt}$ | $k_{d,opt}$ | $k_{i,opt}$              | $k_{d,opt}$             | Min (Min-ITAE) |
|---------|-----------|-------------|-------------|--------------------------|-------------------------|----------------|
| 1       | $k_i-k_d$ | 0.6         |             | $-1.133 \times 10^{-11}$ | 0.712                   | 2.538          |
|         | $k_p-k_i$ |             | 0.5         | 0.470                    | $3.338 \times 10^{-07}$ | 2.703          |
| 2       | $k_i-k_d$ | 2.000       |             | $-1.808 \times 10^{-09}$ | 2.278                   | 0.5762         |
|         | $k_p-k_i$ |             | 2.000       | 1.935                    | $2.013 \times 10^{-12}$ | 0.5184         |
| 3       | $k_i-k_d$ | 1.5         |             | 0.485                    | 1.113                   | 2.398          |
|         | $k_p-k_i$ |             | 1.000       | 1.410                    | 0.460                   | 2.564          |

**Table 2.** TDPMs for different examples and planes

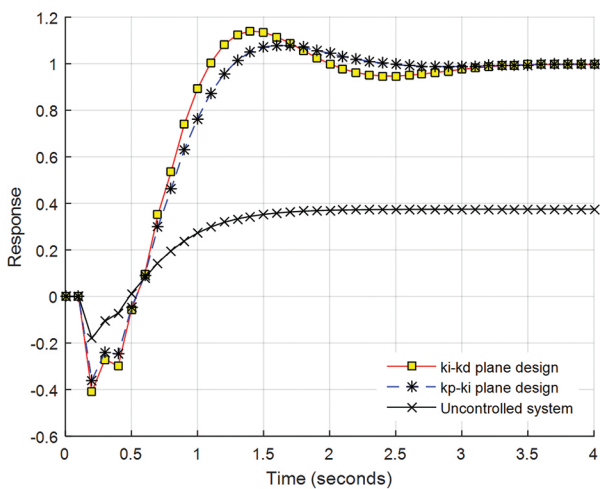
| Example | Plane     | $T_r$ (sec.) | %OS    | %US    | $T_p$ (sec.) | $T_s$ (sec.) | ess   |
|---------|-----------|--------------|--------|--------|--------------|--------------|-------|
| 1       | $k_i-k_d$ | 1.293        | 10.222 | 0      | 3.522        | 6.720        | 0.000 |
|         | $k_p-k_i$ | 2.067        | 1.092  | 0      | 5.083        | 3.935        | 0.000 |
| 2       | $k_i-k_d$ | 0.437        | 14.048 | 41.009 | 1.421        | 2.940        | 0.000 |
|         | $k_p-k_i$ | 0.549        | 7.926  | 36.000 | 1.618        | 2.136        | 0.000 |
| 3       | $k_i-k_d$ | 1.485        | 1.874  | 27.812 | 3.985        | 3.202        | 0.000 |
|         | $k_p-k_i$ | 1.736        | 0.948  | 25.000 | 4.570        | 3.066        | 0.000 |



**Fig. 17.** The unit step response using overall optimum gains for Example 1



**Fig. 19.** The unit step response using overall optimum gains for Example 3.



**Fig. 18.** The unit step response using overall optimum gains for Example 2

It was observed that irrespective of the type and order of the system to be controlled, the  $k_i-k_d$  design plane yielded a faster response while the  $k_p-k_i$  design plane yielded a response that closely tracks the input. From these results, the system design plane can be decided by control system designers since system design specifications depend on its applications.

**5. CONCLUSIONS**

The parametric effects of design plane selection on the performance of PID-controlled time-delay systems were presented in this study. System stability region in the  $k_i-k_d$  and  $k_p-k_i$  planes were determined for a fixed value of  $k_p$  and  $k_d$  respectively. The study concluded that systems required controllers because their closed-loop response can never track the reference input. The controllers were designed by optimising the controller



gains using a combination of GA and graphical methods to get the overall optimum gains.

It was also concluded, that irrespective of the type and order of the system, the  $k_r-k_d$  design plane yielded a system with a faster response while the  $k_p-k_i$  design plane yielded a system response that closely track the input. Also, for a system with right-hand-side zero the  $k_r-k_d$  design plane results in a system response with a higher %US compared to the  $k_p-k_i$  design plane.

The study contributes to aiding the control system designers to select the design plane which gives the best performance right from the start of controller design without using a trial and error approach, once the system transfer function and design specifications are known.

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